



About National Engineering Industries Ltd. (NBC Bearings)

A symbol of dependability and flexible engineering solutions, NBC Bearings is the brand of National Engineering Industries. Founded in 1946, National Engineering Industries Ltd (NEI) is India's leading bearings manufacturer and exporter, renowned for excellence in quality and delivery. In 2021, NBC bearings completed 75 years of its incorporation.

Headquartered in Jaipur, Having started with 30,000 bearings in 19 sizes in 1946, NBC has evolved to manufacture over 200 million bearings each year offering in 2300+ variants to serve a host of customers in India and over 30 other countries across five continents in automotive, railways and industrial segments. NBC also serves the Indian aftermarket through a countrywide network of 550+ authorized stockists and thousands of retailers.

Award & Recognitions:

NBC has been the recipient of several award and accolades for its quality consciousness and manufacturing prowess. Most prominent being the coveted Deming Grand Prize which is the highest honour in quality awarded to a company for excellence in Total Quality Management (TQM). NBC bearings is the only bearing manufacturer to win both - The Deming Application Award and The Deming Grand Prize Award.

The award is given by the Japanese Union of Scientists and Engineers (JUSE) to companies for demonstrating practicing TQM in the areas of production, customer service, safety, human resource, corporate social responsibility, environment, etc. NBC stands committed to an endless journey of continuous improvement through TQM.

Bearing Internal

Type of Clearance During Operation

Internal clearance of a bearing is an important factor affecting not only the bearing performance but also the proper functioning of a machine. Bearing internal clearance is defined as the relative movement of either rings in radial or axial direction, when one ring is fixed. Movement in the diametrical direction is radial clearance, while movement in the shaft's direction is axial clearance. Internal clearance is critical to bearing performance for many reasons. The amount of clearance influences the load distribution in a bearing, which in turn affects smooth operation. It also influences bearing noise and vibration. There are three types of clearance present in the bearing during operation.

Initial clearance: The clearance present inside the bearing before it is mounted on a shaft or housing.

Mounted clearance: The clearance in the bearing after mounting but before the bearing comes into operation.

Operating clearance: The clearance remaining in the bearing after temperature affect and mounting.

For satisfactorily performance bearings must have the appropriate operating clearance. If sufficient amount of clearance is not present in the bearing it may fail. For calculating clearance, effect of fits and temperature is considered. The selection of the clearance is dependent upon the application. In some cases negative clearance (preload) is required when stiffness or bearing positioning is important. Bearing internal clearances changes due to:

- Thermal expansion or contraction of shaft or housing.
- Elastic deformation of rings under load
- Axial clamping can influence clearance or preload.
- Misalignment during running

To get the accurate measurement of internal clearance a certain 'measured load' has to be applied on the raceways. However, under this 'measured load' a slight elastic deformation of the bearing occurs due to which the measured internal clearance value will be slight greater than the true clearance value. This difference between the bearing's true clearance and the measured clearance under the load must be compensated. These compensation values are given in Table 6.1.

Table 6.1. Adjustment of Radial clearance for DGBB

	re Diameter mm	Measuring Load N { Kgf }		Adjustment of internal clearance (Unit µm)						
over	incl.	IN	/ ugi /	C2	CN	C3	C4	C5		
10	18	24.5	{ 2.5 }	3 ~ 4	4	4	4	4		
18	50	49	{5}	4~5	5	6	6	6		
50	200	147	{15}	6~8	8	9	9	9		

Note: For roller bearings the amount of elastic deformation is small enough, to be ignored.

Radial clearance of the bearing is built up for following reasons:

- Accommodate the reduction of clearance in a bearing due to interference for inner ring on the shaft or outer ring in the housing.
- Accommodate the minor changes in the dimensions of parts without affecting the bearing performance.
- Compensate for the differential expansion of the two rings when the inner ring of a bearing operates at a higher temperature than the outer ring.
- It allows a slight misalignment between the shaft and the housing and thereby prevents the premature failure of the bearing.
- It affects the end play of radial ball bearing, and also affects their capacity for carrying axial loads, the greater the radial clearance the greater the capacity for supporting axial load.



Important: Once ball and roller bearings are mounted and running, a small amount of radial internal or running clearance is normally desirable. In the case of bearings under radial load, quieter running is generally obtained when this clearance is minimum.

Types of Radial Internal Clearance:

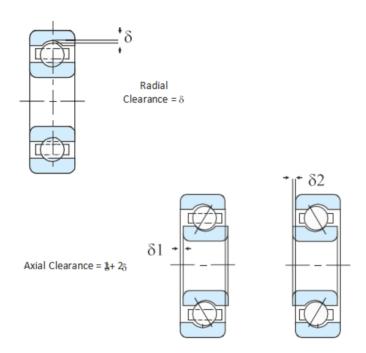
Radial bearings are made with following different ranges of radial internal clearance- C2, Normal, C3 and C4

C2: These bearings have the smallest amount of radial internal clearance. They should only be used where freedom is required in the assembled bearings and there is no possibility of the initial radial internal clearances being eliminated by external causes. Therefore, special attention must be given to the seating dimensions as the expansion of the inner ring or contraction of the outer ring may cause tight bearings. In this respect a C2 bearing should not be used unless recommended.

CN: This grade of radial internal clearance is intended for use where only when one ring is made an interference fit and there is no appreciable loss of clearance due to temperature difference. ball and roller bearings for general engineering applications are usually of this clearance.

C3: This grade of radial internal clearance should be used when both rings of a bearing are made an interference fit or when only one ring is an interference fit but there is likely to be some loss of clearance due to temperature differences. It is the grade normally used for radial ball bearings that take axial loading but for some purposes even bearings with C4 clearance may be required.

C4: Where there will be some loss of clearance due to temperature differences and both rings are interference fit, this grade of radial internal clearance is employed. One example of its use is in bearings for traction motors. Customers should always consult us before ordering bearings with this grade of radial internal clearance.



Criteria For Selection of Internal Clearance

The internal clearance of a bearing under operating conditions (effective clearance) is usually smaller than the same bearing's initial clearance before being installed and operated. This is due to several factors including bearing fit, the difference in temperature between the inner and outer rings, etc. As a bearing's operating clearance has an effect on bearing life, heat generation, vibration, noise, etc. Care must be exercised in selecting the most suitable operating clearance.



Effective Internal Clearance:

The internal clearance differential between the initial clearance and the operating (effective) clearance (the amount of clearance reduction caused by interference fits, or clearance variation due to the temperature difference between the inner and outer rings) can be calculated by the following formula:

 $\delta eff = \delta o - (\delta f + \delta t)$

Where,

δ eff: Effective internal clearance, mm

δo: Bearing internal clearance, mm

 δf : Reduced amount of clearance due to interference, mm

Reduced clearance due to interference:

After installation of bearings with interference fit on shaft and housing, the inner ring will expand and the outer ring will contract; thus reducing the bearings' internal clearance. The amount of expansion or contraction will depend on the shape of the bearing, the shape of the shaft or housing, dimensions of the respective parts, and the type of materials used. The differential can range from approximately 70% to 90% of the effective interference.

 $\delta f = (0.70^{\circ}0.90) \Delta deff$

Where,

 δf : Reduced amount of clearance due to interference, mm

Δ deff: Effective interference, mm



Reduced internal clearance due to inner/outer ring temperature difference.

In operation, normally the outer ring will be 5°to 10°C cooler than the inner ring or rotating parts. However, if the cooling effect of the housing is large, the shaft is connected to a heat source, or a heated substance is conducted through the hollow shaft; the temperature difference between the two rings can be even greater. The amount of internal clearance is thus further reduced by the differential expansion of the two rings as given in the formula:

 $\delta t = \alpha.\Delta T.Do$

Where,

δt: Amount of reduced clearance due to heat differential, mm

 α : Bearing material expansion coefficient 12.5 × 10⁻⁶/°C

ΔT: Inner/outer ring temperature differential, °C

Do: Outer ring raceway diameter, mm

Outer ring raceway diameter, Do, values can be approximated by using formula

For ball bearings and spherical roller bearings,

Do = 0.20 (d + 4.0D)

For roller bearings (except spherical roller bearing),

Do = 0.25 (d + 3.0D)

Where,

d: Bearing bore diameter, mm

D: Bearing outside diameter, mm



Radial Internal Clearance Values

Bearing Internal Clearance Values As Per ISO: 5753/IS: 5935 6.4.1Deep Groove Ball Bearings (Cylindrical bore)

Table 6.2 Radial internal clearance for Deep groove ball bearing with cylindrical bore.





6.4.2(A) Cylindrical Roller Bearing (Interchangeable)

Table 6.3 Radial internal clearance for cylindrical roller bearing (interchangeable) with cylindrical bore.



Clearance values in microns

	ameter d m)	G	roup 2 (C2)		oup N (CN)		oup 3 C3)		up4 '4)	Grou (C:	
Over	Incl.	Min.	Max.	Min.	Max.	Min.	Max.	Min.	Max.	Min.	Max.
-	10	0	25	50	45	35	50	50	75		
10	24	0	25	50	45	35	50	50	75	- 55	90
24	30	0	25	50	45	35	50	50	75	70	95
30	40	5	30	25	50	45	70	60	85	80	105
40	50	5	35	30	60	50	80	70	100	95	125
50	65	10	40	40	70	60	90	80	110	110	140
65	80	10	45	40	75	65	100	90	125	130	165
80	100	15	50	50	85	75	110	105	140	155	190
100	120	15	55	50	90	85	125	125	165	180	220
120	140	15	60	50	105	100	145	145	190	200	245
140	160	20	70	70	120	115	165	165	215	225	275
160	180	25	75	75	125	150	170	170	220	250	300
180	200	35	90	90	145	140	195	195	250	275	330
200	225	45	105	105	165	150	220	220	280	305	365
225	250	45	110	110	175	170	235	235	300	330	395
250	280	55	125	125	195	190	260	260	330	370	440
280	315	55	130	130	205	200	275	275	350	410	485
315	355	65	145	145	225	225	305	305	385	455	535
355	40C	100	190	190	280	280	370	370	460	510	600
400	450	110	210	210	310	310	410	410	510	565	665
45C	500	110	220	220	330	330	440	440	550	525	735



6.4.2(B.1) Cylindrical Roller Bearings (Non-interchangeable)

Table 6.4 Radial internal clearance for Cylindrical roller bearing (non-interchangeable) with cylindrical bore.



Clearance values in microns

Nom in: diamet (mi	ter d	(INA	C	2NA	ı	NA	C3	NA	C41	NA	C5	iNA
Over	Incl.	Min.	Max.	Min.	Max.	Min.	Max.	Min.	Max.	Min. Max.		Min.	Max.
10 18 24	10 18 24 30	5 5 5	10 10 10	10 10	20 20 20 25	20 20 20 25	30 30 35	35 35 35 40	45 45 45 50	45 45 45 50	55 55 55	65 65 70	- 75 75 80
30	40	5	12	12	25	25	40	45	55	55	70	80	95
40	50		15	15	30	30	45	50	65	65	80	95	110
50	65	5	15	15	35	35	50	55	75	75	90	110	130
65	80	10	20	20	40	40	60	70	90	90	110	130	150
80	100	10	25	25	45	45	70	80	105	105	125	155	180
100	120	10	25	25	50	50	80	95	120	120	145	180	205
120	140	15	30	30	60	60	90	105	135	135	160	200	230
140	150	15	35	35	65	65	100	115	150	150	180	225	260
160	180	15	35	35	75	75	110	125	165	165	200	250	285
180	200	20	40	40	80	80	120	140	180	180	220	275	315
200	225	20	45	45	90	90	135	155	200	200	240	305	350
225	250	25	50	50	100	100	150	170	215	215	265	330	380
250	280	25	55	55	110	110	165	185	240	240	295	370	420
280	315	30	60	60	120	120	180	205	265	265	325	410	470
315	355	30	65	65	135	135	200	225	295	295	360	455	520
355	400	35	75	75	150	150	225	255	330	330	405	510	585
400	450	45	85	85	170	170	255	285	370	370	455	565	650

Note: For bearing with normal clearance, NA is added to bearing number, Ex. NU305NA



6.4.2(B.2) Cylindrical Roller Bearings (Non-interchangeable)

Table 6.5 Radial internal clearance for Cylindrical roller bearing (non-interchangeable) with cylindrical bore.



Clearance values in microns

dian	al Bore neter nm)	C9)NA	cc	ONA	C1	NA	C21	NA		NA	C3	NA
Over	Incl.	Min.	Max.	Min.	Max.	Min.	Max.	Min.	Max.	Min.	Max.	Min.	Max.
24 30 40 50	10 18 24 30 40 50	5555555	5 10 10 10 12 15	7 7 7 10 10 10	17 17 17 20 20 20	10 10 10 10 12 15	20 20 20 25 25 30 35	20 20 20 25 25 25 30 35 40	30 30 30 35 40 45	35 35 35 40 45 50 55	45 45 45 50 55 65 75	45 45 45 50 55 65 75	55 55 55 60 70 80
65 80	100	10	20 25	15 20	30 35	20	40 45	45	60 70	80	105	105	110 125
100 120 140 160 180 200 225 250 280	120 140 160 180 200 225 250 280 315	10 15 15 15 20 20 25 25 30	25 30 35 35 40 45 50 55 60	30 30 35 40 40 45	35 40 45 45 50 55 65 65	25 30 35 35 40 45 50 55 60	50 60 65 75 80 90 10 110	50 60 65 75 80 90 100 110	80 90 100 110 120 105 150 165 180	95 105 115 120 140 155 170 185 205	120 135 150 165 180 200 210 240 265	120 135 150 165 180 200 215 240 265	145 160 180 200 220 240 265 295 325
315 355 400 450	355 400 450 500	30 35 45 50	65 75 85 95	45 50 60 70	75 90 10 115	65 75 85 95	135 150 170 190	135 150 170 190	200 225 255 285	225 255 285 315	295 330 370 410	295 330 370 410	360 405 455 505

Note: C9NA, CONA and CINA are applied only to precision bearing of class 5 or higher



6.4.3 Double Row Self Aligning Ball Bearing (Cylindrical bore)

Table 6.6(A) Radial internal clearance for Double row self-aligning ball bearing with cylindrical bore.



Clearance values in microns

Bore di (m		Group 2 (C2)		Group N (CN)		Group 3 (C3)		Group 4 (C4)		Group 5 (C5)	
Over	Incl.	Min.	Max.								
2.5	6	-1	8	5	15	10	20	15	25	21	33
6	10	2	9	6	17	12	25	19	33	27	42
10	14	2	10	6	19	13	26	21	35	30	48
14	18	3	12	8	21	15	28	23	37	32	50
18	24	4	14	10	23	17	30	25	39	34	52
24	30	5	16	11	24	19	35	29	46	40	58
30	40	6	18	13	29	23	40	34	53	46	66
40	50	6	19	14	31	25	44	37	57	50	71
50	65	7	21	16	36	30	50	45	69	62	88
65	80	8	24	18	40	35	60	54	83	76	108
80	100	9	27	22	48	42	70	64	96	89	124
100	120	10	31	25	56	50	83	75	114	105	145
120	140	10	38	30	68	60	100	90	135	125	175
140	160	15	44	35	80	70	120	110	161	150	210

 Table 6.6(B) Radial internal clearance for Double Row Self Aligning Ball

 Bearing with tapered bore.
 Clearance values in microns

	Bore diameter d Group 2 (C2)		Group N (CN)		Group 3 (C3)		Group 4 (C4)		Group 5 (C5)		
Over	Incl.	Min.	Max.	Min.	Max.	Min.	Max.	Min.	Max.	Min.	Max.
18	24	7	17	13	26	20	33	28	42	37	55
24	30	9	20	15	28	23	39	33	50	44	62
30	40	12	24	19	35	29	46	40	59	52	72
40	50	14	27	22	39	33	52	45	65	58	79
50	65	18	32	27	47	41	61	56	80	73	99
65	80	23	39	35	57	50	75	69	98	91	123
80	100	29	47	42	68	62	90	84	116	109	144
100	120	35	56	50	81	75	108	100	139	130	170
120	140	40	68	60	98	90	130	120	165	155	205
140	160	45	74	65	110	100	150	140	191	180	240



6.4.4 (A) Spherical Roller Bearing (Cylindrical bore)

Table 6.7(A) Radial internal clearance for Spherical roller bearing with cylindrical bore.



Clearance values in microns

Nominal bord diameter		Bear	ing with cylindri	cal bore	
d mm	C2	CN	C3	C4	CS
er Incl	min max	min max	min max	min max	min max
14 18	10 20	20 35	35 45	45 60	60 75
18 24	10 20	20 35	35 45	45 60	60 75
24 30	15 25	25 40	40 55	55 75	75 95
30 40	15 30	30 45	45 60	60 80	80 100
40 50	20 35	35 55	55 75	75 100	100 125
50 65	20 40	40 65	65 90	90 120	120 150
65 80	30 50	50 80	80 110	110 145	145 180
80 100	35 60	60 100	100 135	135 180	180 225
100 120	40 75	75 120	120 160	160 210	210 26
120 140	50 95	95 145	145 190	190 240	240 300
140 160	60 110	110 170	170 220	220 280	280 350
160 180	65 120	120 180	180 240	240 310	310 390
180 200	70 130	130 200	200 260	260 340	340 430
200 225	80 140	140 220	220 290	290 380	380 470
225 250	90 150	150 240	240 320	320 420	420 520
250 280	100 170	170 280	260 350	350 460	460 570
280 315	110 190	190 280	280 370	370 500	500 630
315 355	120 200	200 310	310 410	410 550	550 690
355 400	130 220	220 340	340 450	450 600	600 750
400 450	140 240	240 370	370 500	500 660	660 820
450 500	140 260	260 410	410 550	550 720	720 900
500 560	150 280	280 440	440 600	600 780	780 1,000
560 630	170 310	310 480	480 650	650 850	850 1,100
630 710	190 350	350 530	530 700	700 920	920 1,190
710 800	210 390	390 580	580 770	770 1,010	1,010 1,300
800 900	230 430	430 650	650 860	860 1,120	1,120 1,440
900 1,000	280 480	480 710	710 930	930 1,220	1,220 1,570
1,000 1,120	290 530	530 780	780 1,020	1.020 1,330	1,330 1,720
1,120 1,250	320 580	580 860	860 1,120	1,120 1,480	1,460 1,870
1,250 1,400	350 640	640 950	950 1,240	1,240 1,620	1,620 2,080



6.4.4 (B) Spherical Roller Bearing (Cylindrical bore)

Table 6.7(B) Spherical Roller Bearing (Tapered bore)



Clearance values in microns

	iameter d m)	Gro	up 2 (2)	Grou (C	up N N)	Group 3 (C3)		Group 4 (C4)			up 5 5)
Over	Incl.	Min.	Max.	Min.	Max.	Min.	Max.	Min.	Max.	Min.	Max.
18	24	15	25	25	35	35	45	45	60	60	75
24	30	20	30	30	40	40	55	55	75	75	95
30	40	25	35	35	50	50	65	65	85	85	105
40	50	30	45	45	60	60	80	80	100	100	130
50	65	40	55	55	75	75	95	95	120	120	160
65	80	50	70	70	95	95	120	120	150	150	200
80	100	55	80	80	110	110	140	140	180	180	230
100	120	65	100	100	135	135	170	170	220	220	280
120	140	80	120	120	160	160	200	200	260	260	330
140	160	90	130	130	180	180	230	230	300	300	380
160	180	100	140	140	200	200	260	260	340	340	430
180	200	110	160	160	220	220	290	290	370	370	470
200	225	120	180	180	250	250	320	320	410	410	520
225	250	140	200	200	270	270	350	350	450	450	570
250	280	150	220	220	300	300	390	390	490	490	620
280	315	170	240	240	330	330	430	430	540	540	680
315	355	190	270	270	360	360	470	470	590	590	740
355	400	210	300	300	400	400	520	520	650	650	820
400	450	230	330	330	450	450	570	570	720	720	910
450	500	260	270	270	490	490	630	630	790	790	1000
500	560	290	410	410	540	540	680	680	870	870	1100
560	630	320	460	460	600	600	760	760	980	980	1230
630	710	350	510	510	670	670	850	850	1090	1090	1360
710	800	390	370	370	750	750	960	960	1220	1220	1500
800	900	440	640	640	840	840	1070	1070	1370	1370	1690
900	1000	490	710	710	930	930	1190	1190	1520	1520	1860



Preload

Bearing Preload is the process of applying an axial load, independent of external loads, to a bearing. An axial preload ensures constant contact between the rolling element and raceways reducing or eliminating play inside bearing.

Bearings are preloaded to increase rigidity and thereby to reduce unwanted bearing's displacement. This means that the bearings' internal clearance is negative before operation. This is called "preload" and is commonly applied to angular ball bearings and tapered roller bearings.

The amount of preload applied is important. It must be sufficient to reduce the excess play, but care must be taken not to apply too much preload. Proper preload allows the rolling elements to freely rotate in the bearing races, while excessive preload could lead to skidding. This will increase friction and heat generation, which can ultimately lead to premature bearing failure.

Purpose of preload

- Bearing's rigidity increases, internal clearance tends not to be produced even when heavy load is applied.
- (2) The particular frequency of the bearing increases and is becomes suitable for high-speed rotation.
- Shaft run-out is suppressed; rotation and position precision are enhanced.
- (4) Vibration and noise are controlled.
- (5) Sliding of rolling elements by turning, spinning, or pivoting, is controlled and smearing is reduced.
- (6) Fretting produced by external vibration is prevented.

Preloading different types of bearings

1. Deep groove ball bearings

Deep groove ball bearings requiring a preload must be loaded axially. Axial load can be applied to the outer ring or inner ring. The race comes in contact with the ball and translates this load to the raceway of the outer or inner ring. This creates a contact angle, between the race, the ball and the outer race. Preloading of deep groove ball bearings is optional but



preloading of angular contact bearings is mandatory. Preloading can be done using a constant-force adjustable spacer (spring washer). Spring preloading reduces noise and is widely used in electric motors

2. Angular contact bearing

Angular contact bearings have large contact angles typically 15°, 25°, 30° or 40°. Angular contact bearings which are manufactured in matched pairs are referred to as duplex bearings. Duplex bearings are manufactured with the bearing faces adjacent to each other in face to face or back to back arrangement. The raceways of duplex bearings are slightly offset but

come together when the proper preload is applied during assembly.

In Back-to-back arrangement, the bearings has a clearance between the inner ring faces. After preloading, the inner rings are clamped together, bringing the inner ring and outer ring faces flush with each other and resulting in contact angles that diverge towards the centerline.







In Face-to-face bearings assembly, there is a clearance between the outer rings. After preloading, the outer ring faces and inner ring faces are clamped together, bringing the faces flush with each other and resulting in contact angles that converge towards the centerline.

Tandem assembly two or more bearings are arranged in the same orientation. During assembly, the inner rings are clamped together, resulting in contact angles that are parallel to each other and increases the thrust capacity in one direction only.



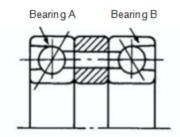


Preload methods

There are two main methods of preloading bearings

Rigid preloading

It provides the higher radial and axial rigidity. The preload is obtained by applying a load with a precision locknut or clamping element and a spacers is used between the two bearings. The nut applies an axial load to the inner or outer bearing ring and thus preloads bearings. Shims may also be used to obtain the proper preload





Spring preload

Also referred as constant pressure preloading. Spring is used for applying an axial force to the non-rotating inner or outer of bearing. The force applied by the spring is relatively constant throughout its compression. Spring preloading can withstand high operating speeds as compared to rigid preloading.

